1. Objectives

The objective in this experiment is to design a state-feedback controller for the rotary inverted pendulum module using the LQR technique. The controller will maintain the pendulum in the inverted (upright) position and should be robust in order to maintain its stability in the case of a disturbance.

Upon completion of the exercise, you should have experience in the following:

- How to mathematically model the rotary inverted pendulum system.
- To linearize the model about an equilibrium point.
- To use the LQR method in designing a state-feedback controller.
- To design and simulate a WinCon controller for the system.
- To use feedback to stabilize an unstable system.

2. System Requirements

To complete this Lab, the following hardware is required:

[1] Quanser UPM 2405/1503 Power Module or equivalent.
[1] Quanser MultiQ PCI / MQ3 or equivalent data acquisition card.
[1] Quanser SRV02 servo plant.
[1] PC equipped with the required software as stated in the WinCon user manual.

- The required configuration of this experiment is the SRV02 in the High-Gear configuration along with a ROTPEN – Rotary Pendulum Module as well as a UPM 2405/1503 power module and a suggested gain cable of 1.

- It is assumed that the student has successfully completed Experiment #0 of the SRV02 and is familiar in using WinCon to control the plant through Simulink.

- It is also assumed that all the sensors and actuators are connected as per dictated in the SRV02 User Manual and the Rotary Pendulum User Manual.
3. Mathematical Model

Figure 1 below depicts the Rotary inverted pendulum module coupled to the SRV02 plant in the correct configuration. The Module is attached to the SRV02 load gear by two thumbscrews. The Pendulum Arm is attached to the module body by a set screw. The Inverted Pendulum experiment is a classical example of how the use of control may be employed to stabilize an inherently unstable system. The Inverted pendulum is also an accurate model in the pitch and yaw of a rocket in flight and can be used as a benchmark for many control methodologies.

The following table is a list of the nomenclature used in the following illustrations and derivations.

<table>
<thead>
<tr>
<th>Symbol</th>
<th>Description</th>
<th>Symbol</th>
<th>Description</th>
</tr>
</thead>
<tbody>
<tr>
<td>$L$</td>
<td>Length to Pendulum's Center of mass</td>
<td>$h$</td>
<td>Distance of Pendulum Center of mass from ground</td>
</tr>
<tr>
<td>$m$</td>
<td>Mass of Pendulum Arm</td>
<td>$J_{cm}$</td>
<td>Pendulum Inertia about its center of mass</td>
</tr>
<tr>
<td>$r$</td>
<td>Rotating Arm Length</td>
<td>$V_x$</td>
<td>Velocity of Pendulum Center of mass in the x-direction</td>
</tr>
<tr>
<td>$\theta$</td>
<td>Servo load gear angle (radians)</td>
<td>$V_y$</td>
<td>Velocity of Pendulum Center of mass in the y-direction</td>
</tr>
<tr>
<td>$\alpha$</td>
<td>Pendulum Arm Deflection (radians)</td>
<td></td>
<td></td>
</tr>
</tbody>
</table>
Figure 2 above depicts the rotary inverted pendulum in motion. Take note of the direction the arm is moving. Figure 3 depicts the pendulum as a lump mass at half the length of the pendulum. The arm is displaced with a given $\alpha$. Notice that the direction of $\theta$ is now in the $x$-direction of this illustration. We shall begin the derivation by examining the velocity of the pendulum center of mass.

Referring back to Figure 3, we notice that there are 2 components for the velocity of the Pendulum lumped mass:

$$V_{\text{Pendulum center of mass}} = -L \cos \alpha (\dot{x}) - L \sin \alpha (\dot{\alpha}) \dot{y}$$  \[3.1\]

We also know that the pendulum arm is also moving with the rotating arm at a rate of:

$$V_{\text{arm}} = r \dot{\theta}$$  \[3.2\]

Using equations [3.1] & [3.2] and solving for the $x$ & $y$ velocity components:

$$V_x = r \theta - L \cos \alpha (\dot{x})$$
$$V_y = -L \sin \alpha (\dot{\alpha})$$  \[3.3\]

Equation [3.3] leaves us with the complete velocity of the pendulum. We can now proceed to derive the system dynamic equations.
3.1 Deriving The System Dynamic Equations

Now that we have obtained the velocities of the pendulum, the system dynamic equations can be obtained using the Euler-Lagrange formulation. We obtain the Potential and Kinetic energies in our system as:

**Potential Energy** - The only potential energy in the system is gravity:

\[
V = P.E._{\text{Pendulum}} = mgh = mgL\cos\alpha \tag{3.4}
\]

**Kinetic Energy** - The Kinetic Energies in the system arise from the moving hub, the velocity of the point mass in the x-direction, the velocity of the point mass in the y-direction and the rotating pendulum about its center of mass:

\[
T = K.E._{\text{Hub}} + K.E._{v_x} + K.E._{v_y} + K.E._{\text{Pendulum}} \tag{3.5}
\]

*Note: Since we have modeled the pendulum as a point mass at its center of mass, the total kinetic energy of the pendulum is the kinetic energy of the point mass plus the kinetic energy of the pendulum rotating about its center of mass.*

The moment of inertia of a rod about its center of mass is:

\[
J_{cm} = \frac{1}{12}MR^2
\]

since we’ve defined \( L \) to be half the pendulum length, then \( R \) in this case would be equal to \( 2L \). Therefore the moment of inertia of the pendulum about its center of mass is:

\[
J_{cm} = \frac{1}{12}MR^2 = \frac{1}{12}M(2L)^2 = \frac{1}{3}ML^2 \tag{3.6}
\]

Finally, our complete kinetic energy \( T \) can be written as:

\[
T = \frac{1}{2}J_{eq}\dot{\theta}^2 + \frac{1}{2}m(r\dot{\theta} - L\cos\alpha(\dot{\alpha}))^2 + \frac{1}{2}m(-L\sin\alpha(\dot{\alpha}))^2 + \frac{1}{2}J_{cm}\dot{\alpha}^2 \tag{3.7}
\]

After expanding equation [3.7] and collecting terms, we can formulate the Lagrangian:

\[
L = T - V = \frac{1}{2}J_{eq}\dot{\theta}^2 + \frac{2}{3}mL^2\dot{\alpha}^2 - mLr\cos\alpha(\dot{\theta}) + \frac{1}{2}m \dot{\theta}^2 - mgL\cos\alpha \tag{3.8}
\]
Our 2 generalized co-ordinates are $\theta$ and $\alpha$. We therefore have 2 equations:

$$\frac{\delta}{\delta t} \left( \frac{\delta L}{\delta \theta} \right) - \frac{\delta L}{\delta \theta} = T_{\text{output}} - B_{eq} \dot{\theta}$$  \[3.9\]

$$\frac{\delta}{\delta t} \left( \frac{\delta L}{\delta \alpha} \right) - \frac{\delta L}{\delta \alpha} = 0$$  \[3.10\]

Solving Equations \[3.9\] & \[3.10\] and linearizing about $\alpha = 0$, we are left with:

$$(J_{eq} + mr^2) \ddot{\theta} - mLr \ddot{\alpha} = T_{\text{output}} - B_{eq} \dot{\theta}$$  \[3.11\]

$$\frac{4}{3} mL^2 \ddot{\alpha} - mLr \ddot{\theta} - mgL \alpha = 0$$  \[3.12\]

Referring back to Experiment #1 – Position Control, we know that the output Torque on the load from the motor is:

$$T_{\text{output}} = \frac{\eta_m \eta_g K_i K_g (V_m - K_g K_m \theta)}{R_m}$$  \[3.13\]

Finally, by combining equations \[3.11\], \[3.12\] & \[3.13\], we are left with the following state-space representation of the complete system:

$$\begin{bmatrix} \dot{\theta} \\ \dot{\alpha} \\ \ddot{\theta} \\ \ddot{\alpha} \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & bd & -cG & 0 \\ 0 & ad & -bG & 0 \end{bmatrix} \begin{bmatrix} \theta \\ \alpha \\ \dot{\theta} \\ \dot{\alpha} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ c \frac{\eta_m \eta_g K_i K_g}{R_m E} \\ b \frac{\eta_m \eta_g K_i K_g}{R_m E} \end{bmatrix} V_m$$

Where:

$a = J_{eq} + mr^2$

$E = ac - b^2$

$G = \frac{\eta_m \eta_g K_i K_g^2 + B_{eq} R_m}{R_m}$

$c = \frac{4}{3} mL^2$

$d = mgL$
In the typical configuration of the SRV02 & the ROTPEN (Pendulum/inverted pendulum) system, the above state space representation of the system is:

\[
\begin{bmatrix}
\dot{\theta} \\
\dot{\alpha} \\
\ddot{\theta} \\
\ddot{\alpha}
\end{bmatrix} =
\begin{bmatrix}
0 & 0 & 1 & 0 \\
0 & 0 & 0 & 1 \\
0 & 39.32 & -14.52 & 0 \\
0 & 81.78 & -13.98 & 0
\end{bmatrix}
\begin{bmatrix}
\theta \\
\alpha \\
\dot{\theta} \\
\dot{\alpha}
\end{bmatrix} +
\begin{bmatrix}
0 \\
0 \\
25.54 \\
24.59
\end{bmatrix} V_m
\]

3.2 Pre Lab Assignment

The purpose of the lab is to design a state-feedback controller that will balance the inverted pendulum in the upright position. The controller should also maintain the pendulum’s stability (upright) while allowing the user to command varying setpoints of the servo angle.

The controller specifications for this lab are:

- The controller must maintain system stability.
- The controller must accurately place the servo angle (theta) at a given command while maintaining the pendulum in the upright position.
- The pendulum angle (α) must not exceed ±6° when the system is excited with a step input to the θ command.
- The control signal (V_m) must be strictly bounded by ±2.5 Volts.
- The settling times of θ & α should not exceed 2 s each for a step input (θ command of 1 rad).

The first task upon entering the laboratory will be to simulate the non-linear model of the rotary inverted pendulum and the above state-space linear model. The goal of this simulation is to determine the limits of the linear model (what range of α does the linear model accurately describe the motion of the pendulum?).

The following equations below describe the complete non-linear system. The linear system is derived by linearizing these two equations about α = 0. The pre-lab assignment is to determine a rough estimate of the range of α that the linear model will hold true.

\[
a \ddot{\theta} - b \cos(\alpha) \ddot{\alpha} + b \sin(\alpha) \dot{\alpha}^2 + G \dot{\theta} = \frac{\eta_m \eta_g K_i K_g}{R_m} V_m
\]

\[
c \ddot{\alpha} - b \cos(\alpha) \ddot{\theta} - d \sin(\alpha) = 0
\]
4. In Lab Procedure

The Rotary inverted pendulum is an ideal experiment when introducing important controls concepts such as disturbance rejection and non-linear systems. The intent of this laboratory is to first validate the use of the linear model (through simulation) and then proceed to design a closed-loop full state feedback controller that will maintain the pendulum in its upright position. The inverted pendulum has many practical applications in that it can be used to model many non-linear systems.

The purpose of the lab is to design a state-feedback controller that will balance the inverted pendulum in the upright position. The controller should also maintain the pendulum's stability (upright) while allowing the user to command varying setpoints of the servo angle.

The controller specifications for this lab are:

- The controller must maintain system stability.
- The controller must accurately place the servo angle (theta) at a given command while maintaining the pendulum in the upright position.
- The pendulum angle (\( \alpha \)) must not exceed \( \pm 6^\circ \) when the system is excited with a step input to the \( \theta \) command.
- The control signal (\( V_m \)) must be strictly bounded by \( \pm 2.5 \) Volts.
- The settling times of \( \theta \) & \( \alpha \) should not exceed 2 sec each for a step input (\( \theta \) command of 1 rad).

4.1 Part I - Verification of the Linear Model

The first part of this lab will be verifying our linear model by simulating the linear and non-linear systems together and looking for the point of divergence. This exercise will not only verify our linear model, but will also establish a threshold to turn the controller off when \( \alpha \) exceeds the threshold.

The first task upon entering the laboratory is to familiarize yourself with the system. The pendulum deflection signal (\( \alpha \)) should be connected to encoder channel #1 and the servomotor's position signal (\( \theta \)) should be connected to encoder channel #0. Analog Output channel #0 should be connected to the UPM (Amplifier) and from the amplifier to the input of the servomotor. This system has one input (\( V_m \)) and two outputs (\( \theta \ & \alpha \)).

You are now ready to begin the lab. Launch MATLAB from the computer connected to the system. Under the “SRV02_Exp7_Inverted Pendulum” directory, begin by running the file by the name “Setup_SRV02_Exp7.m”. This MATLAB script file will setup all the specific system parameters and will set the system state-space matrices A,B,C & D. You are now ready verify the linear model.

Under the same directory, open a Simulink model called “s_SRV02_Non_Linear_Pendulum.mdl”. This model is a simulation of the complete non-linear pendulum system
as well as the linearized state-space representation of the system. The purpose of this simulation is to investigate the validity of the linear model and also to determine the range of \( \alpha \) where the linear model correctly describe the system.

Start the simulation. You should have 2 scopes open that are displaying alpha (\( \alpha \)) and theta (\( \theta \)). To put the plots in perspective, \( \alpha \) was given an initial condition slightly greater than 0 and thus allowed to fall. Figure 4 Below is a plot of the linear and non_linear simulations of \( \alpha \).

![Simulation of Non-Linear & Linear Models](image)

As you can seen from the above simulation, the linear model correctly depicts the motion of the pendulum for the first 1.5 seconds and then begins to break down. In the simulation plot of alpha, zoom into graph between 1.4 & 1.6 seconds. Here you will see the two curves begin to diverge. It seems that the linear model quite accurately describes the system for the first 25° and then begins to diverge from the actual motion. We will therefore set the Alpha_Threshold variable to 25°. It should be noted that a range of ±25° is quite acceptable for this application as the controller you will be designing will try to maintain the pendulum in the upright position (\( \alpha = 0 \)).

### 4.2 Part II – Designing & Simulating the Controller

Now that we have verified the linear model to be accurate in the range of motion we are expecting, you will now use this linear model to design a full-state feedback controller to maintain the pendulum in the upright position and to meet the controller specifications mentioned above. The method of calculating the feedback gains will be the LQR function in MATLAB’s control systems toolbox.

The MATLAB LQR function returns a set of calculated gains based on the system matrices A & B and the design matrices Q & R. In this section of the Lab, you will begin the iterative design process by varying Q and taking notice on the effect those changes have on the simulated system response.
For the purpose of this lab, we will fix the Q matrix to only be comprised of 2 elements (q₁,q₂) and we will hold R at 1. The reason behind limiting the design to only 2 variables is to investigate the role of emphasis in an LQR design. q₁ is mainly associated with θ while q₂ is mainly associated with α. With this in mind, the user can select which variable the controller will put more emphasis on forcing to 0. *(The result of any stable full-state feedback controller will be to drive all states to 0).*

\[
Q = \begin{bmatrix}
q_1 & 0 & 0 & 0 \\
0 & q_2 & 0 & 0 \\
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{bmatrix}
\]

The above Q matrix is the simple design matrix you will be using for this controller. In actuality, the user may vary all the elements of the Q matrix to obtain the optimal controller, but this is not necessary for the purpose of this lab.

Under the same directory, open a model called “s_SRV02_Inverted_Pendulum.mdl”. This model is a simulation of the Linear model with a feedback law \( u = -kx \). The gain vector \( k \) is set by the LQR function. Before beginning the simulation, open a Matlab script file called “Iteration_RotPen.m”. Keep this script open as you will be executing it repetitively. You may save & run this file by pressing the F5 key. Run this script and then proceed to the Simulink model and start the simulation. You should have 3 scopes open that are displaying alpha (α), theta (θ) and the control signal (u). Every time the “Iteration_RotPen.m” script runs, you will see the new response in the simulation as well as two step-responses (θ & α) in MATLAB figures. If you right-click on these figures and choose Characteristics->Settling Time, the settling-time of the step response will be shown.

The default Q values are \( Q = \text{diag} ([1 1 0 0]) \) as you can see in the Iteration script. At this point, you should start varying q₁ & q₂ and making note of their effect on the closed loop system characteristics (θ settling time, α settling time, α range, u range). By the end of this exercise you should have a table of the form:

<table>
<thead>
<tr>
<th>q₁</th>
<th>q₂</th>
<th>θ (θ) Settling time (s)</th>
<th>α (α) Settling time (s)</th>
<th>α Range</th>
<th>u Range</th>
</tr>
</thead>
<tbody>
<tr>
<td>1</td>
<td>1</td>
<td>2.55</td>
<td>2.65</td>
<td>-3.2° &lt; α &lt; 3.2°</td>
<td>-1 V &lt; u &lt; 1 V</td>
</tr>
<tr>
<td>10</td>
<td>1</td>
<td>1.1</td>
<td>1.51</td>
<td>-7.2° &lt; α &lt; 7.2°</td>
<td>-3.2 V &lt; u &lt; 3.2 V</td>
</tr>
<tr>
<td>1</td>
<td>10</td>
<td>2.55</td>
<td>2.67</td>
<td>-2.9° &lt; α &lt; 2.9°</td>
<td>-1 V &lt; u &lt; 1 V</td>
</tr>
</tbody>
</table>

*Table 1 - Iteration Table : Sample Entries*

Table 1 above shows 3 different entries into the Iteration Table that you should be constructing. In total there should be 20 entries in the table (vary each parameter in 10 steps while holding the other parameter at 1). Suggested ranges for this exercise are:

\[ 0 < q₁ < 20 ; \quad 0 < q₂ < 50 \]
Now that your **Iteration Table** is complete, look back at the controller requirements of this lab while cross-referencing your newly formed table to determine what parameters would result in meeting the specifications.

Once you determine the best set of parameters that meet the specified requirements, you are ready to test your controller on the actual plant. Make sure you have properly documented how you obtained your final set of parameters in a table much like the **Iteration Table** and clearly show how the requirements have been met. Before closing the "Iteration_RotPen.m" file, make sure you run it one last time with your final design parameters ($q_1$ & $q_2$) such that the gain vector $k$ is set accordingly. You may now stop the simulation.

### 4.3 Part III - Implementing the Controller

After verifying the calculated controller gains, it is time to implement the controllers on the actual system. In the same working directory, open a Simulink model called "q_SRV02_Inverted_Pendulum.mdl". This model has the I/O connection blocks linking to the physical plant as well as a simulated block to compare real and simulated results. You may now proceed to "**Build**" the controller through the WinCon menu. Before starting the controller, make sure that the pendulum rod is in its correct staring position. The starting position of the pendulum should match the setting given in the experiment setup file “Setup_SRV02_Exp7.m”. Open this file and scroll to the following section:

```matlab
% ################################ USER-DEFINED PENDULUM CONFIGURATION ################################
% Pendulum Start Position: set to 'UP' or 'DOWN'
PENDULUM_START = 'UP';
% ################################ END OF USER-DEFINED PENDULUM CONFIGURATION ################################
```

![Figure 5 - PENDULUM_START = 'DOWN'

![Figure 6 - PENDULUM_START = 'UP']
Referring to Figure 5 & Figure 6, please make sure that the pendulum arm is in the correct starting location as is set in the “Setup_SRV02_Exp7.m” file. If the pendulum setup you are using is not in the same configuration as set in the setup file, you should notify the lab technician or T.A. to correct the error. Now that the pendulum arm is in the correct starting position as set in the setup file, you are ready to start the controller.

*Note: Figure 6 shows the PENDULUM_START in the 'UP' position. Notice how the pendulum arm is sitting flush on the right stopper. Figure 5 shows the PENDULUM_START in the 'DOWN' position. Notice how the pendulum arm is fastened to the tip of the shaft.

Start the controller through WinCon. The controller is now running but the plant is not moving as you must first enable the system by having the pendulum arm cross the \( \alpha = 0 \) threshold. With one finger on the tip of the pendulum, slowly move the pendulum toward the upright position until you feel the system begin to work. This controller was designed to remain dormant until the pendulum arm has crossed the upright position and enabled the controller. Your controller should now be running.

*WARNING: If at any point the system is not behaving as expected, make sure to immediately press STOP on the WinCon server.

*If at any time you hear a high frequency 'hum' from the system, this is an indication that the gains are too high and you need to re-calculate your controller.

Through the WinCon server, open a plot of 3 signals (Measured Alpha, Measured Theta & Control Signal). You should be noticing the position of the servo (Theta) is oscillating. This is referred to as a Limit Cycle and is cause by the static friction in the system. A more thorough investigation of this phenomenon is out of the scope of this laboratory.

You must now ensure that you controller meets the required specifications. Create a table similar to the Measurement Table below. Switch back to the Simulink diagram and enter a Theta Setpoint of 28° (28° is approximately equal to 0.5 radians). Due to the Limit Cycle, it is necessary to gather measurements of 5 step inputs and compute the average. As your controller is sending a step-input into the system, you should be filling out your Measurement Table. It is recommend to change the buffer of the WinCon plots to 10 seconds as to get 2 full cycles of data per plot.

<table>
<thead>
<tr>
<th>Sample #</th>
<th>Alpha Range</th>
<th>Control Signal (u) Range</th>
</tr>
</thead>
<tbody>
<tr>
<td>1</td>
<td>(-4.5^\circ &lt; \alpha &lt; 5.5^\circ)</td>
<td>(-2 \text{ V} &lt; u &lt; 2 \text{ V})</td>
</tr>
<tr>
<td>..</td>
<td>..</td>
<td>..</td>
</tr>
<tr>
<td>5</td>
<td>(-4.7^\circ &lt; \alpha &lt; 5.3^\circ)</td>
<td>(-2.2 \text{ V} &lt; u &lt; 2.2 \text{ V})</td>
</tr>
<tr>
<td>Average</td>
<td>(-4.63^\circ &lt; \alpha &lt; 5.4^\circ)</td>
<td>(-2.05 \text{ V} &lt; u &lt; 2.1 \text{ V})</td>
</tr>
</tbody>
</table>

Table 2 - Measurement Table - Sample Entries
Using the average as an indication of your overall controller performance, does your controller still meet the system requirements? (Due to the Limit Cycle, the Alpha & Theta settling time measurements would not give an actual reading and thus not be an accurate indication of controller performance. If your controller did not manage to meet the specified requirements, you should go back to the simulations and re-iterate the design procedure until you have developed a controller that will meet the requirements. The previous figures show the WinCon plots used to make the above calculations. It should be noted that each sample in the Measurement Table corresponds to one 10-second capture of the plots as seen in Figure 7 & Figure 8 above. You should also capture a plot of Theta as it is required to answer some of the post-lab questions.

Figure 7 - Alpha Signal Capture

Figure 8 - Control Signal Capture
5. Post Lab Question and Report

Upon completion of the lab, you should begin by documenting your work into a lab report. Included in this report should be the following:

i. In the pre-lab, you were asked to theorize as to the range of alpha that you would expect the linear model to be valid. Include all calculations and the final estimate.

ii. In Part II of the lab, you were asked to vary 2 parameters (10 steps for each) for a total of 20 entries. Make sure to include your Iteration Table in this report.

iii. With the controller requirements in mind, you were asked to determine the optimal parameters to achieve those specifications. Include your design steps and all iterations used in determining the final controller.

iv. After implementing your designed controller on the real plant in Part III of the lab, you were asked to compile a Measurement Table. Make sure this is included in your final report.

v. In Part III, you implemented your controller on the physical plant. Comment on the performance of your controller on the actual system as opposed to the simulated model.

vi. Make sure to include your final controller gains and any re-iterative calculations made if any.

5.1 Post Lab Questions

1) Having performed this lab by using the LQR technique, what other control approaches would you consider for this system?

2) The inverted pendulum is a classical example of a non-minimum phase system. In what way is this true. Prove your argument with plots from the actual plant.

3) By how much did your measurements of the actual controller deviate from those of the simulated model? Give some quantitative measurements and comment on what you perceive as being the source of these discrepancies.

4) Once you implemented your controller on the actual system, you should have noticed the Limit Cycle phenomenon. What ways would there be of incorporating this into your model so that it would show up in the simulations.